

GAMME COMPLÈTE DE GRADES ET DE FINITIONS • RAPIDITÉ DE LIVRAISON • ISO 9001 VASTE INVENTAIRE • SERVICE DE FABRICATION SUR MESURE • SYSTÈME DE TRAÇABILITÉ



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Register of International Marks



Self-Locking Nut

HARDLOCK NUT INTRODUCTION

FEATURES OF HARDLOCK NUT

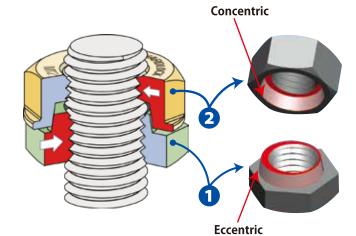
- ◆ Reusable without reduction in performance!
- ◆ Full torque management and completely fastened even with ZERO (0) clamp load!
- ◆ Available in various materials and surface treatments tailored to the environment!
- ◆ No special tools required for installation!

LOCKING MECHANISM

HARDLOCK NUT consists of two nuts, the first nut "Convex Nut" ①(clamping nut) has a truncated protrusion arranged off-center on the upper surface.

The second nut "Concave Nut" ②(locking nut) is designed with a concentric conical recess for locking the two nuts together.

By tightening the concave nut onto the convex nut, a strong perpendicular load will be applied to the bolt from both sides.

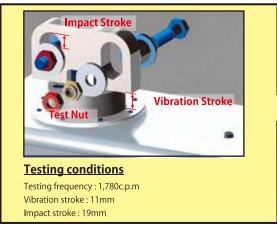


PROVED SUPERIOR IN A VARIETY OF LOOSENING TESTS

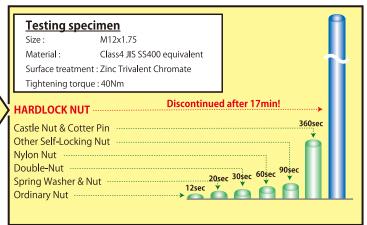
1) Accelerated vibration test conforming to NAS 3350/3354 (National Aerospace Standard)

To determine the capability of fasteners to withstand accelerated vibration condition.

Assembly of NAS testing machine



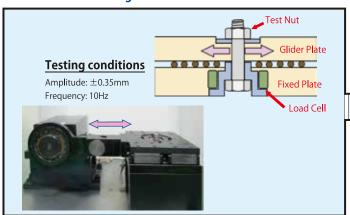




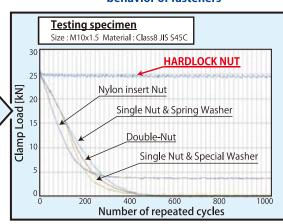
2) Junker Vibration Test

The test bench applies a transverse cyclic vibration to the glider plate, and the clamp load is measured in real time and plotted on a graph.

Section through of Junker test machine



Comparison of the self-loosening behavior of fasteners



CASE EXAMPLE OF COST REDUCTION

Although initial cost is higher than standard bolted connection, HARDLOCK Nut provides significant reduction in total maintenance cost by eliminating re-tightening work with resultant decrease in maintenance frequency and time.

Flywheel fastening for VIBRATING SCREEN (Mining company in Brazil)

Vibrating screens are used in mining industry to separate feeds containing solids and crushed rocks/ores by force of vibration generated from a flywheel, consequently always being exposed to severe vibration.





Pendulum flywheel installed with ordinary nuts





Vibrating Screen

Reduction of yearly Maintenance cost (per 1 machine)

M24×3.0 C8	
48 pcs / machine	
	Initial Cost
	(Rough estimate)
Ordinary Nut	US\$ 57.6
Ordinary Nat	US\$1.2×48 pcs
HARDLOCK Nut	US\$ 240
TIANDLOCK NAC	US\$5×48 pcs
Cost Difference	+US\$ 182.4

	Maintenance Cost												
(a)	(b)	(c)	$(d)=(a)\times(b)\times(c)$	(e)	$(f)=(d)\times(e)$								
Engineers	Maintenance hours	Maintenance per year	Man-Hour per year	Labor Cost (/hour)	Maintenance Cost per year								
5 (Engineers)	4 (hours)	12 (times)	240(MHRS)	US\$ 20	US\$ 4,800								
5 (Engineers)	1 (hour)	1 (time)	5(MHRS)	US\$ 20	US\$ 100								
					-US\$ 4,700								

Maintenance cost reduced by up to 98%

Railway joint applications (Railway company in Japan)

Two rails are bolted to join together on a track. A force as strong as 500G is applied when a train passes, this causes heavy stress on the joint as well as strain from expansion and contraction of the rail.









Installed with HARDLOCK nuts

HARDLOCK nut can produce significant locking effect even with a low torque of 250 - 300Nm, compared to 500Nm for an ordinary nut. This also contributes to decrease of bolt breakage by creating a fine balance in the rail joint as if it allows the joint to "breathe" and relieves it from stress.

Reduction of yearly Maintenance cost (for 500 joints)

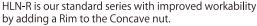
M20×2.5 C8 w/bolt & washer									
500 joints × 2 rails × 4 sets									
4,000 sets	Initial Cost								
	(Rough estimate)								
Ordinary Nut	US\$ 18,000 US\$4.5×4,000 pcs								
HARDLOCK Nut	US\$ 32,000 US\$8×4,000 pcs								
Cost Difference	+US\$ 14,000								

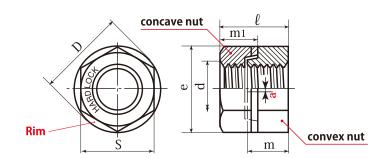
	neduction of yearly maintenance cost (101 500 joints)													
	Maintenance Cost													
(a)	(b)	$(c)=(b) \times 12$	$(d)=(a)\times(c)$	(e)	$(f)=(d)\times(e)$									
Engineers	Maintenance per month	Maintenance per year	Man-day per year	Labor Cost (/day)	Maintenance Cost per year									
3 (Engineers)	3 (days)	36 (days)	108(man-day)	US\$ 300	US\$ 32,400									
2 (Engineers)	1 (day)	12 (days)	24(man-day)	US\$ 300	US\$ 7,200									
					-US\$ 25,200									

HARDLOCK NUT DIMENSION TABLE1

HLN-R: RIM TYPE







by adding a	a Rim to	the Concave	nut.						U	Init : mm	
Nominal size	Со	Thick nvex nut		cave nut		Width oss flats	e	Overall height	Rim dia.	Unit weight	Recommended tightening torque for the concave nut
	m		m1		S			e	D	(g)	(N-m)
d	Basic	Tolerance	Basic	Tolerance	Basic	Tolerance	approx.	approx.	approx.	approx.	Common to all (Min - Max)
M5×0.8	4	0.1 -0.15	4	0.5 -0.2	8	0 -0.2	9.2	7.2	9.2	1.9	2 - 3
M6×1.0	5	±0.3	5	0 -0.3	10	0 -0.6	11.5	8.5	11.5	4	4 - 5
M8×1.25	6.5	0 -0.58	6.5	0 -0.58	13	0 -0.7	15.0	10.8	15.0	8.9	9 - 13
M10×1.5	8	0 -0.58	8	0 -0.58	17	0 -0.7	19.6	13.2	19.6	18	18 - 24
M12×1.75	10	0 -0.58	9.3	0 -0.58	19	0 -0.8	21.9	16.0	21.9	26	27 - 39
M16×2.0	13	±0.9	11	0 -0.7	24	0 -0.8	27.7	21.2	27.7	46	70 - 100
M20×2.5	16	±0.9	14.5	0 -0.7	30	0 -0.8	34.6	26.7	34.6	93	120 - 200
M22×2.5	18	±0.9	15.6	0 -1.2	32	0 -1	37.0	29.9	37.0	115	150 - 250
M24×3.0	19	±0.9	17.6	0 -1.2	36	0 -1	41.6	32.4	41.6	183	160 - 300
M27×3.0	21	±1.0	17.6	0 -1.2	41	0 -1	47.3	33.5	47.3	243	250 - 390
M30×3.5	23	±1.0	18.6	0 -1.2	46	0 -1	53.1	36.5	53.1	312	270 - 440

External dimensions: JIS B1181(2004) / ISO 4302(Width across flats only)

Screw thread tolerances: JIS B0209(2001) / ISO 965 6H

AVAILABLE STEEL GRADE:

AVAILA	<u> </u>	LLL OI	<u> </u>				
Strength Class	Clas	ss 4	Class 8	Class 10	A2-70		
Steel Grade	Low cark	oon steel	Medium carbon steel	Chromium Molybdenum steel	Stainless steel 304		
Applicable	JIS SS400 e	equivalent	JIS S45C	JIS SCM435	JIS SUS304 equivalent		
Standard	ASTM A563 G	ir. A equivalent	ASTM A194 Gr. 2H equivalent	ASTM A194 Gr. 7 equivalent	ASTM A194 Gr. 8 equivalent		
Surface finish	Zinc plating trivalent chromate	Hot Dip Galvanized (HDZ35)	Manganese Phosphate	Manganese Phosphate	_		
M5×0.8	✓	_	_	_	✓		
M6×1.0	✓	_	_	_	✓		
M8×1.25	✓	✓	✓	√	✓		
M10×1.5	✓	✓	✓	√	✓		
M12×1.75	✓	✓	✓	✓	✓		
M16×2.0	✓	✓	✓	✓	✓		
M20×2.5	✓	✓	✓	✓	_		
M22×2.5	✓	✓	✓	√	_		
M24×3.0	✓	✓	✓	✓	_		
M27×3.0	✓	✓	✓	✓	_		
M30×3.5	✓	✓	✓	✓	_		

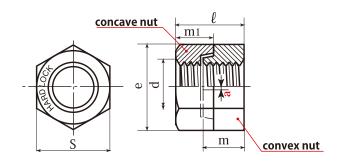
[•] In the case of HDZ, please tighten the concave nut 50% more than the above torque value due to the high torque coefficient.

HARDLOCK NUT DIMENSION TABLE2

HLN: BASIC TYPE



HLN-B is the original series of HARDLOCK NUT.



Unit:mm

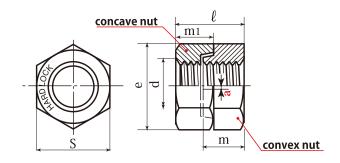
Nominal	Pite	ch		Thick	ness		1	Width		Overall	Unit	Recommended	
size			Со	nvex nut		cave nut	acr	oss flats	е	height	weight	tightening torque for the concave nut	
d	p Coarse		Basic	m Tolerance	Basic	m1 Tolerance	Basic	S Tolerance	approx.	(g) approx.	approx.	(N-m) Common to all (Min - Max)	
M6	1.0	0.75	5	±0.48	5	±0.48	10	0 -0.6	11.5	9.2	3.3	4 - 5	
M8	1.25	1.0	6.5	±0.58	6.5	±0.58	13	0 -0.7	15.0	12.0	8.6	9 - 13	
M10	1.5	1.25	8	±0.58	8	±0.58	17	0 -0.7	19.6	14.4	17.6	18 - 24	
M12	1.75	1.25	10	±0.58	10	±0.58	19	0 -0.8	21.9	17.9	27.3	27 - 39	
M14	2.0	1.5	11	±0.7	11	±0.7	22	0 -0.8	25.4	19.9	39	40 - 58	
M16	2.0	1.5	13	±0.9	12	±1.0	24	0 -0.8	27.7	23.2	52.8	70 - 100	
M18	2.5	1.5	15	±0.9	14	±1.0	27	0 -0.8	31.2	26.7	80	100 - 150	
M20	2.5	1.5	16	±0.9	15	±1.0	30	0 -0.8	34.6	28.2	105	120 - 200	
M22	2.5	1.5	18	±0.9	17	±1.0	32	0 -1	37.0	32.3	130	150 - 250	
M24	3.0	2.0	19	±0.9	18	±1.0	36	0 -1	41.6	33.9	180	160 - 300	
M27	3.0	2.0	21	±1.0	21	±1.0	41	0 -1	47.3	37.9	246	250 - 390	
M30	3.5	2.0	23	±1.0	23	±1.0	46	0 -1	53.1	41.9	375	270 - 440	
M33	3.5	2.0	25	±1.0	20	0 -1.5	50	0 -1	57.7	39.4	411	290 - 490	
M36	4.0	3.0	28	±1.0	21	0 -1.5	55	0 -1	63.5	41.9	532	340 - 590	
M39	4.0	3.0	30	±1.2	23	0 -1.5	60	0 -1.2	69.3	45.7	681	390 - 640	
M42	4.5	4.0	33	±1.2	25	0 -1.5	65	0 -1.2	75.0	50.2	892	440 - 690	
M45	4.5	4.0	35	±1.2	27	0 -1.5	70	0 -1.2	80.8	54.2	1,115	490 - 740	
M48	5.0	4.0	37	±1.2	29	0 -1.5	75	0 -1.2	86.5	58.2	1,393	540 - 780	
M52	5.0	4.0	41	±1.2	31	0 -1.5	80	0 -1.2	92.4	63.7	1,708	590 ~ 830	
M56	5.5	4.0	44	±1.2	34	0 -1.5	85	0 -1.4	98.1	68.7	2,047	640 ~ 880	
M64	6.0	4.0	50	±1.5	38	0 -1.5	95	0 -1.4	110	77.0	2,795	690 ~ 930	

Screw thread tolerances: JIS B0209(2001) / ISO 965 6H

HARDLOCK NUT DIMENSION TABLE3

HLN-B: BASIC TYPE INCH THREAD SERIES





Unit:inch

Nominal Size	Conv	ex nut	Conca	Concave nut		ross flats	е	Overall Unit height weight		Recommended tightening torque for the concave nut	
-Threads per inch	n	n	m1		!	5		e	(g)	(N-m)	
	Max.	Min.	Max.	Min.	Max.	Min.	approx.	approx.	approx.	Min - Max	
1/4-20 UNC	0.226	0.212	0.226	0.212	0.438	0.428	0.488	0.390	3.3	4 - 5	
5/16-18 UNC	0.273	0.258	0.273	0.258	0.500	0.489	0.557	0.457	7.9	9 - 13	
3/8-16 UNC	0.337	0.320	0.337	0.320	0.562	0.551	0.628	0.559	17.6	18 - 24	
7/16-14 UNC	0.385	0.365	0.385	0.365	0.688	0.675	0.768	0.638	20.8	27 - 39	
1/2-13 UNC	0.448	0.427	0.448	0.427	0.750	0.736	0.840	0.752	28.1	40 - 58	
5/8-11 UNC	0.559	0.515	0.559	0.515	0.938	0.922	1.051	0.972	52.8	70 - 100	
3/4-10 UNC	0.665	0.597	0.665	0.597	1.125	1.088	1.240	1.165	105	120 - 200	
7/8-9 UNC	0.776	0.704	0.776	0.704	1.312	1.269	1.447	1.370	130	150 - 250	
1-8 UNC	0.887	0.811	0.887	0.811	1.500	1.450	1.653	1.567	246	200 - 350	
1 1/8-7 UNC	0.999	0.919	0.999	0.919	1.688	1.631	1.859	1.776	310	260 - 420	
1 1/4-7 UNC	1.094	1.010	0.751	0.667	1.875	1.812	2.066	1.583	324	280 - 470	
1 3/8-6 UNC	1.206	1.118	0.815	0.727	2.062	1.994	2.273	1.728	436	320 - 550	
1 1/2-6 UNC	1.317	1.225	0.880	0.788	2.250	2.175	2.480	1.843	551	370 - 620	
1 3/4-5 UNC	1.540	1.440	1.009	0.909	2.625	2.538	2.893	2.189	896	470 - 720	
2-4.5 UNC	1.763	1.655	1.138	1.030	3.000	2.900	3.306	2.433	1,363	570 - 800	

Dimensions...ASME/ANSI B18.2.2 1987(R1999) Thread Requirements...ANSI B1.1a-1968 2B

INSTALLATION PROCEDURE:









- 1 Use a tightening tool (spanner, torque wrench etc.) to tighten the Convex Nut to the appropriate torque for the application. The Convex Nut has the same Strength Class as a regular hexagon nut and can therefore be tightened to its maximum limit.
- 2 Install the Concave nut onto the Convex nut by hand until it no longer turns. Prior to tightening of the Concave nut, make sure that there is about 1 thread pitch gap between the nuts. If not, please refrain from using HARDLOCK NUT with the current bolt.

 If the space is narrower than that of 1 thread pitch, the nut would be unable to demonstrate sufficient locking effect. The same conditions apply to reuse
- 3 Use a torque wrench to tighten the Concave nut to the recommended torque shown in this catalog.
- 4 Even after tightening the nuts correctly, there is a chance that there is a small gap between the nuts due to the tolerance of bolt diameter. However, even with or without a space, if tightened correctly as instructed in this installation procedure, the HARDLOCK NUT will produce sufficient locking effect.

CASE EXAMPLE OF IMPROVEMENT

Railway (Cars)







Before

- In high speed rail cars, repeated impact of the rail coupling causes the nut to loosen and may even risk detachment.
- Damage to the friction ring also causes the nut to loosen.

After

- By using the HARDLOCK Nut, even under repeated impact, loosening is prevented and detachment has not occurred.
- Breakage of the friction ring is eliminated by using the HARDLOCK Nut and has been adopted by many rail car manufacturers.

Construction Equipment < Hydraulic Breaker>





Before

 Hydraulic breakers used at large mines and quarries used double nuts fastened with high torque. Severe vibration caused loosening on a daily basis.

After

- HARDLOCK Nut was used to overcome nut loosening at the appropriate tightening torque.
- The HARDLOCK Nut works with standard thread froms, and therefore no bolt specification changes were needed so the HARDLOCK Nut could be implemented quickly.

Railway < Turnout >







Before

- When a train passes, a force as large as 500G is applied to the switch, and even specially shaped double nuts used on the movable parts can become loose on a daily basis.
- While the switch (movable part) is essential to maintain the track gauge, it is extremely difficult to maintain the gauge of the track while preventing loosening.

After

- By using the HARDLOCK Nut, the maintenance cost associated with inspection and re-tightening was successfully reduced.
- The HARDLOCK Nut successfully maintained the gauge of the track and prevents loosening simultaneously. As a result, the Hardlock Nut has been adopted by all railway companies in Japan

Wind Power <Tower>



Before

- Wind turbine towers stand in areas of strong wind, due to irregular stresses caused by strong wind load, double nuts and spring washers are frequently used to prevent loosening.
- Nylon nuts are used to fasten the cable rack wiring inside the tower and the ends of the ladder, but loosening still occured, due to the micro vibrations caused by the blade rotation.

After

- By using the HARDLOCK Nut, the correct torque and bolt load could be managed at each of the fastening points.
- As a result, the frequency of routine maintenance was extended and the costs were significantly reduced.

Electrical Equipment <Switchboard Terminal Fastener>



Before

• Steps to prevent loosening had been used including double nuts and the use of spring washers but vibrations during transportation and micro vibrations that occur after installation had resulted in frequent loosening.

After

- After switching to the HARDLOCK Nut, all problems related to loosening were completely solved.
- Because loosening does not occur, both the number of inspections and re-tightening work has been significantly reduced resulting in labor savings.

With HardLock Innovation, You Find a Distinctive Solution

SARDLOCK® One of the control of the	‡	NCRD-LCCK°
Excellent clamp load retention	£	Good clamp load retention
Can be used multiple times with consistant results	×	Can only be used once for consistant results
Does not interfere with base material or its finish		"Bites" into base material to insure proper grip
Can be used from 0 lbf to designed clamp load		Can only be used at designed clamp load
Installed only on the nut side of the bolt assembly	Secured Secured Bolt can rotate to loosen Secured	Needs to be installed on both the nut side and bolt head side to reach good vibration resistance

GENERAL TIGHTENING TORQUE FOR HEXAGON NUT

		Stress	Str	ength C	lass	Clamp	' Hantening tordile			Clamp Tightening torque				Clamp Tightening torque			, raus
Size	Pitch	Area	В	olt		load	rignt	ening to	orque	load	rignt	ening to	orque	load	rignt	ening to	orque
		2		YP	Nut	YP×0.7	K=0.15	K=0.2	K=0.25	YP×0.8	K=0.15	K=0.2	K=0.25	YP×0.9	K=0.15	K=0.2	K=0.25
mm	mm	mm²	20	N/mm²	61 0	kN	0	N·m	4.4	kN		N·m	4.5	kN	4.0	N·m	47
M6	1.0	20	8.8	640	Class8	9	8	11	14	10	9	12	15	12	10	14	17
			10.9	900	Class10 Class8	13 16	11 20	15 26	19 33	15 19	13 22	17 30	22 37	16 21	15 25	20 34	24 42
M8	1.25	37	8.8	900	Class10	23	28	37	46	26	32	42	53	30	36	47	59
			8.8	640	Class8	26	39	52	65	30	45	59	74	33	50	67	84
M10	1.5	58	10.9	900	Class10	37	55	73	91	42	63	84	105	47	70	94	117
			8.8	640	Class8	38	68	91	113	43	78	104	130	49	87	117	146
M12	1.75	84	10.9	900	Class10	53	96	127	159	61	109	146	182	68	123	164	205
			8.8	640	Class8	52	108	144	180	59	124	165	206	66	139	185	232
M14	2.0	115	10.9	900	Class10	73	152	203	254	83	174	232	290	93	196	261	326
			8.8	640	Class8	70	169	225	281	80	193	257	322	90	217	289	362
M16	2.0	157	10.9	900	Class10	99	237	316	396	113	271	362	452	127	305	407	509
		400	8.8	640	Class8	86	232	310	387	98	265	354	442	111	299	398	498
M18	2.5	192	10.9	900	Class10	121	327	436	545	138	373	498	622	156	420	560	700
1420	2.5	245	8.8	640	Class8	110	329	439	549	125	376	502	627	141	423	564	706
M20	2.5	245	10.9	900	Class10	154	463	618	772	176	529	706	882	198	595	794	992
M22	2.5	303	8.8	640	Class8	136	448	597	746	155	512	682	853	175	576	768	960
IVIZZ	2.5	303	10.9	900	Class10	191	630	840	1,050	218	720	960	1,200	245	810	1,080	1,350
M24	3.0	353	8.8	640	Class8	158	569	759	949	181	651	867	1,084	203	732	976	1,220
1712-4	3.0	333	10.9	900	Class10	222	801	1,068	1,334	254	915	1,220	1,525	286	1,029	1,372	1,716
M27	3.0	459	8.8	640	Class8	206	833	1,110	1,388	235	952	1,269	1,586	264	1,071	1,428	1,785
14127	3.0	133	10.9	900	Class10	289	1,171	1,562	1,952	331	1,339	1,785	2,231	372	1,506	2,008	2,510
M30	3.5	561	8.8	640	Class8	251	1,131	1,508	1,885	287	1,292	1,723	2,154	323	1,454	1,939	2,424
			10.9	900	Class10	353	1,590	2,120	2,651	404	1,818	2,423	3,029	454	2,045	2,726	3,408
M33	3.5	694	8.8	640	Class8	311	1,539	2,052	2,565	355	1,759	2,345	2,931	400	1,979	2,638	3,298
			10.9	900	Class10	437	2,164	2,886	3,607	500	2,474	3,298	4,123	562	2,783	3,710	4,638
M36	4.0	817	8.8	640	Class8	366	1,976	2,635	3,294	418	2,259	3,012	3,765	471	2,541	3,388	4,235
			10.9	900	Class10	515	2,779	3,706	4,632	588	3,176	4,235	5,294	662	3,574	4,765	5,956
M39	4.0	976	8.8	640	Class8	437	2,558	3,410	4,263	500	2,923	3,898	4,872	562	3,289	4,385	5,481
-			10.9	900	Class10	615	3,597	4,796	5,995	703	4,111	5,481	6,851	791	4,625	6,166	7,708
M42	4.5	1,120	8.8	640	Class10	502	3,161	4,215	5,269	573	3,612	4,817	6,021	645	4,064	5,419	6,774
			10.9 8.8	900	Class10 Class8	706 587	4,445 3,962	5,927 5,282	7,409 6,603	806 671	5,080 4,527	6,774 6,036	8,467 7,545	907 755	5,715 5,093	7,620	9,526
M45	4.5	1,310	10.9	900	Classo Class10	825	5,571	7,428	9,285	943	6,367	8,489	10,611	1,061	7,162	6,791 9,550	8,489 11,937
<u> </u>			8.8	640		659	4,742	6,323		753	5,419	-	9,031	847			
M48	5.0	1,470	10.9	900	Class8 Class10	926	6,668	8,891	7,903	1,058	7,620	7,225 10,161	12,701	1,191	6,096 8,573	8,129 11,431	10,161 14,288
			8.8	640	Class 10	789	6,150	8,200	10,251	901	7,020	9,371	11,714	1,014	7,907	10,543	13,179
M52	5.0	1,760	10.9	900	Class10	1,109	8,649	11,532	14,414	1,267	9,884	13,179	16,474	1,426	11,120	14,826	18,533
			8.8	640	Class8	909	7,639	10,185	12,732	1,039	8,731	11,641	14,552	1,169	9,822	13,096	16,370
M56	5.5	2,030	10.9	900	Class10	1,279	10,743	14,324	17,905	1,462	12,277	16,370	20,462	1,644	13,812	18,416	23,020
			8.8	640	Class8	1,201	11,526	15,368	19,210	1,372	13,173	17,564	21,955	1,544	14,819	19,759	24,699
M64	6.0	2,680	10.9	900	Class10	1,688	16,209	21,612	27,014	1,930	18,524	24,699	30,874	2,171	20,840	27,786	34,733
			. 5.5		2.03310	.,500	. 5,207	2.,012		.,550	. 5,521	,555	55,57	_,.,.	20,010	2.,,00	3.11.33

YP : Yield Point K : Torque coefficient

The tensile strength of HARDLOCK Convex nuts are completely the same as regular hexagon nuts. Therefore, the above tightening torque value can be used when tightening the Convex nut.

HARDLOCK Industry Co., Ltd.

1-6-24, Kawamata, Higashi Osaka, Osaka, Japan 577-0063

TEL: +81-6-6784-1131 FAX: +81-6-6784-1161

https://www.hardlock.co.jp/en/ Email: h.office@hardlock.co.jp

Company Profile

to provide unrivaled Safety and Increased Performance



Name HARDLOCK Industry Co., Ltd.

Established April 1, 1974

President Katsuhiko Wakabayashi

Business sites

Main Office: 1-6-24 Kawamata, Higashi-Osaka, Osaka Prefecture 577-0063

Plants: Nos. 1 – 5 (Higashi-Osaka)

Warehouse: Product Center (Higashi-Osaka)

Tokyo Office: 2-5-9 Higashi-Ueno, Taito-ku, Tokyo 110-0015

R&D Center: Rm. 302 Product Development Support Laboratory, Tokyo Metropolitan Industrial

Technology Research Institute: 2-4-10 Aomi, Koto-ku, Tokyo 135-0054

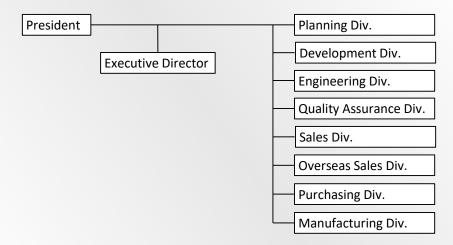
Sales Turnover 2 billion JPY (FY 2017)

Overseas dealers
UK, China, Korea, Czech, Thailand, Brazil, Australia, Taiwan, other
The Bank of Tokyo-Mitsubishi UFJ, Ltd., Sumitomo Mitsui Banking Corp.

Patents and 50

other acquired rights

Organization



Products

HARDLOCK Nut (HLN)



To fasten bolted joint

Size

Standard Rim: M5 to M30

Basic: M6 to M130

Fine Pitch threads, Thin body type

Inch Series

Material

Class 4 JIS SS400 equiv. Class 8 JIS S45C

Class 10 JIS SCM435

A2 IIC CLIC204

A2 JIS SUS304

(Also available in other materials)

HARDLOCK Bearing Nut (HLB)



To set bearing axially on a shaft or on adapter sleeves

Size

M10 to M200

Material

Class 4 JIS SS400 equiv. Class 8 JIS S45C Class 10 JIS SCM435

A2 JIS SUS304

HARDLOCK Set Screw (HLS)





To fix a core pin for die casting die

Size

M8 to M36

Standard Normal Standard Hollow Pin Normal Pin Hollow

Material

Class 8 JIS S45C Class 10 JIS SCM435

History

Make society safer and more secure with fasteners that



1974

Established HARDLOCK Co. Apr in Nagata, Joto-ku, Osaka.

1977

Mar Renamed the company HARDLOCK Co., Lid. (Capitalized at 4 million JPY) Opened a factory in Higashi-June Osaka.

1979 Mar

Began manufacturing and selling the HLB (HARDLOCK Bearing Nut).

1982

Oct Opened the Tokyo Office in Ueno, Taito-ku, Tokyo.

1990

Moved the Head Office and Aug factory to the current location.

1994

Increased capital to 10 Aug million JPY.

1996

May Moved the Tokyo Office to the current location.

1997

Built a third factory on the Apr site of the Head Office.

1998

Received the R&D Award at Jan the 15th Annual Top **Executive Awards from The** Nikkan Kogyo Shimbun.

Dec Received the Excellent Company Award from The Nikkan Kogyo Shimbun and the President's Award from the Higashi-Osaka Chamber of Commerce and Industry.

1991

Jun Began manufacturing and selling to limited customers the SLN (Space Lock Nut).

2000

Began manufacturing and Apr selling to limited customers the HLS (HARDLOCK Set Screw) and HL-P (HARDLOCK Pinlock Bolt).

Jul Officially registered the "HARDLOCK" trademark (Reg. No. 4402041).

Oct Built a fourth factory on the site of the Head Office. Created a Trading Dept.

Dec 2001

May Obtained approval from Railtrack of the UK for the HARDLOCK Nut to be used with railway signals (Authorization No. PA05/867).

2002

May Began manufacturing and selling to limited customers the SLB (Space Lock Bearing Nut).

2003

HARDLOCK Nut was selected Jan for the Good Design Award. It was the first-ever product in the industry to receive the award.

HARDLOCK Nut was adopted for Taiwan's Rapid Transit System.

Oct **HARDLOCK Nut was** required for use in specific locations by Network Rail of the UK (Authorization No. PA05/2077).

Acquired ISO9001-2000 Νον certification from TUV of Germany.

Dec Obtained certification to use the HARDLOCK Nut from Queensland Rail, Australia. (Certificate No. C0054)

2005

Jul Self-locking effect of the HARDLOCK Nut was announced in a research paper at the ASME PVP2005 Conference held in Denver, Colorado (USA).

2006

Dec Effectiveness of the selflocking effect of the HARDLOCK Nut was introduced in a documentary on train accidents by the BBC (UK), leading to wider-spread use in the railway industry.

2007

Mar Selected for a Silver Medal at the 9th Higashi-Osaka Manufacturing Awards.

Listed amongst the Top **Small Japanese Business** Known Worldwide by Newsweek magazine.

2008

Apr Given top scores in delivery, quality and price by Etesia of France.

May Rated top level in loosening tests conducted by the China Academy of Machinery Science and Technology

Jun Expanded the Head Office Factory.

Obtained approval for the Jul HARDLOCK Nut to be used with railway cars by Pesa of Poland. HARDLOCK Nut was adopted for use on the ATR220 for Trenitalia of Italy.

2009

Apr Named amongst the 300 Most Active Small Manufacturing Business by The Small and Medium **Enterprise Agency** lun

Named amongst the Top 100 Manufacturing Businesses in the Kansai by the Kinki Bureau of Economy, Trade and Industry.

Jul Selected for Special Recognition at the 3rd Monozukuri Nippon Grand Awards by the Prime Minister's Office.

Nov Honored with The Order of the Rising Sun, Gold and Silver Rays at the Autumn Commendations by the Emperor Heisei.

2010

Hosted Masayuki Naoshima, Jan Minister of Economy, Trade and Industry, and his delegation for a factory tour.

Received the Invention Mar Award at the 35th Invention Awards from the Japan Society for the Advancement of Inventions and Nikkan Kogyo Shimbun.

2011

Mar President Wakabayashi's "The Screw That Never Comes Loose: How a Small Company Became a Global Player" was published and simultaneously released in China, Korea, Taiwan and elsewhere.

Oct Hosted Motohisa Furukawa, Minister of State for National Policy, for a tour of small businesses.

Nov Received the Japan Innovators Award at the 10th Japan Innovators Awards from Nikkei Business Publications.

2012

June ... President Wakabayashi delivered a special address for the 60th anniversary of his alma mater, the Osaka Institute of Technology.

July ... Hosted Win Myint, Myanmar Minister of Commerce, and his delegation of 25 persons for a tour of exemplar small businesses in Japan. Also, hosted Masajuro Shiokawa, former Minister of Finance.

Aug. ... President Wakabayashi delivered an address at the Summer Employee Seminar of the Ministry of Finance.

2013

Hosted Toshimitsu Motegi, Jan Minister of Economy, Trade and Industry, for a factory tour and opinion-sharing with President Wakabayashi.

Exhibited products in the World Factory Area of The World's Best Laboratory for **Everyone at Grand Front** Osaka.

Jun Registered the HARDLOCK Nut in the New Technology Information System, a database of technologies used in public works projects of the Ministry of Land, Infrastructure, Transport and Tourism (Reg. No. KK-130006-A).

2014

Jul Built the Product Center to commemorate 40 years of business.

Oct Received a letter of appreciation from the President of JR Tokai for contributions to safety, as a part of commemorative activities marking 50 years of service of the Tokai Shinkansen.

2015

Received the Kuninomiya Apr Higashi Memorial Award from the Kuninomiya Higashi Society.

Received the Cool Japan Jun Award from the Cool Japan Association.

Nov Received the Kuninomiya Higashi Culture Award from the Kuninomiya Higashi Society.

2016 Jul

Registered the "HARDLOCK" trademark in China.

2017

Launched joint research into May self-locking fasteners for the medical field in cohort with the Tokyo Metropolitan Industrial Technology Research Institute.

Sep Obtained approval to use the HARDLOCK Nut with fishplates from Network Rail of the UK, leading to use all across the UK.

Dec Launched development of ultra-small HARDLOCK Nuts for the aerospace and medical fields. Built a fifth factory on the site of the Head Office. Conferred HARDLOCK Industry President's Awards to recipients at the WW Idea Competition of the Kuninomiya Higashi Society.

2018

Jan Registered the "HARDLOCK" trademark in Korea.

Customers

Building bonds between people, between things, between everything



<Japan>

<Railway infrastructure & rolling stock>

JR, private railways, subways Kawasaki Heavy Industries Nippon Sharyo Hitachi Kinki Sharyo

Japan Transport Engineering

<Transportation equipment>

Toyota Motor
Nissan Motor
Honda Motor
Mitsubishi Motors
Isuzu Motors
Suzuki Motor
Fuji Heavy Industries
Hino Motors
Yamaha Motor
Nippon Fruehauf
Nippon Trex
TCM

Aichi Kyokuto Kaihatsu Kogyo

<Electric power and gas>

Tokyo Electric Power
Tohoku Electric Power
Chubu Electric Power
Hokuriku Electric Power
Kansai Electric Power
Kyushu Electric Power
Shikoku Electric Power
Chugoku Electric Power
Okinawa Electric Power
Hokkaido Electric Power
Tokyo Gas

<Nuclear power generation>

Fukushima Daiichi NPP 1 - 6 Fukushima Daini NPP 1, 3 Kashiwazakikariwa NPP 1 - 7 Onagawa NPP 1 - 3 Higashidori NPP 1 Hamaoka NPP 1 - 5

Oi PP

Takahama PP Shika NPP Tsuruga PP

Mihama PPSpent Nuclear Fuel Reprocessing Plant

Tokai Daini PP Shimane NPP Ikata PP Genkai NPP Sendai NPP Oarai R&D Institute

Monju NPP Fugen NPP

Ningyotoge Environmental Engineering

Center Toki Office, NIFS Ako Office, Spring-8

JNF Rokkasho Reprocessing

Plant

QST Naka Fusion Institute

<Pylons>

NTT/NTT Docomo

KDDI Civil Aviation Bureau Defense Facilities Administration Bureau National Police Agency Railway related Municipalities

ULVAC

<Cement>

Taiheiyo Cement Ube-Mitsubishi Cement Sumitomo-Osaka Cement <Amusement>

Universal Studios Japan Kompan Playscape

<Industrial equipment>

Kubota Yanmar Ebara Fanuc

Nissei Plastic Industrial Sumitomo Heavy Industries

Toshiba Machine
Japan Steel Works
Brother Industries
Torishima Pump
Manufacturing
DMG Mori
Fujifim
Ulvac

<Aluminum sashes>

Nippon Light Metal Sumitomo Light Metal Industries

Tostem
YKK AP
Shinnikkei
Sankyo Tateyama
Fuji Sash
<Construction>
Kajima
Taisei
Obayashi
Shimizu
Takenaka

<Electric equipment>

Mitsubishi Electric Toshiba Fuji Electric Panasonic Meidensha Hitachi

<Shipvards and plants>

Mitsubishi Heavy Industries Hitachi Zosen Mitsui E&S Kawasaki Heavy Industries Ishikawa-Harima Heavy Industries Sumitomo Heavy Industries

Tsukishima Kikai

Hitachi Construction Machinery

Komatsu

Kobelco Construction

Machinery Tadano Ube Machinery Sakai Heavy Industries

<Steel>

Nippon Steel & Sumitomo Metal

JFE Steel Kobe Steel

<Construction>

Kajima Taisei Obayashi Shimizu Takenaka

<Overseas>

[Korea]

Osaka Gas

Samsung Heavy Industries Hyundai Heavy Industries

Posco (Steel) Hyundai Motor GM-Daewoo KIA Motor

Hyundai Rotem (Rolling stock)

Korea Railroad
Woojin (AGT)
Samsung Electronics
Hanwah Techwin
Ulvac Korea
Incheon Bridge
Korea Electric Power

[India]

ASB International

(Injection molding equipment)

Suzuki Powertrain Vita Technology (Valves) Sesa Goa Iron Ore

[Thailand]

BLCP Power

(Hydroelectric power generation)

[Malaysia]

Gempita Engineering (Valves

and pumps)
[Philippines]
Coral Bay Nickel

[China]

China Railway High-Speed Hong Kong Metro

Guangzhou Metro Bao Steel

[Taiwan]

Taiwan Rapid Transit System

China Steel
Formosa Plastics
Taiwan Railways
Administration
Dragon Steel
[Singapore]

SMRT (Subway) New Transit System

[Pakistan] Bestway (Cement)

[UEA] Dubai New Transit

[Maldives] Huawei Marine [Australia]

Bycyrus (Mining equipment) Hydrapower Pty.Ltd. Queensland Rail (Railway) Westinghouse Rail Systems

Vossloh Cogifer (Rail infrastructure)

[Sweden]

Atlas Copco Volvo Dynapac

[Austria] Hoerbger (Valves)

Plasser & Theurer (Railway

infrastructure)
[Germany]
Siemens

[USA]

Cooper Crane & Rigging Arcelor Mittal U.S.A.

John Deere IHI Press GE Energy New York Metro

[Mexico] Nissan Automotive parts [Brazil]

Vale S.A Arcelor Mittal

Metos Sao Marco Andritz

[UK]

Network Rail

London Underground Balfour Beatty Unipart Rail

SPX Rail

Westinghouse Rail System Morrison (Police Masts) Kee Process (Sewage tanks) Abacus Lighting (Mobile phone

masts)
[France]

Etesia (Lawnmowers)

Vossloh Cogifer (Railway Point)

[Poland]

Pesa (Rolling stock)

[Italy] Ferrari

[Switzerland]

ABB AS (Robotics)